

MULTIMEDIA



UNIVERSITY

STUDENT ID NO

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MULTIMEDIA UNIVERSITY

FINAL EXAMINATION

TRIMESTER 1, 2019/2020

EME1046 – PRINCIPLES OF THERMODYNAMICS
(ME)

24 OCTOBER 2019
9.00 a.m – 11.00 a.m
(2 Hours)

INSTRUCTIONS TO STUDENTS

1. This question paper consists of 5 printed pages (including cover page and appendix) with four questions.
2. Attempt **ALL FOUR** questions. Each question carries 25 marks.
3. Please write all your answers in the Answer Booklet provided.
4. All necessary workings must be shown.
5. A property tables booklet is provided.

Question 1

Air undergoes a thermodynamic cycle consisting of three processes as shown in **Figure Q1**:

Process 1–2: Isothermal expansion from $T_1 = 400$ K, $P_1 = 6$ bar, $V_1 = 0.03$ m³ to $V_2 = 3V_1$

Process 2–3: Constant volume to P_3 where ($P_3 < P_2$)

Process 3–1: Adiabatic compression to state 1.

There are no significant changes in kinetic or potential energy.

(Use $C_v = 0.718$ kJ/kg, $R = 0.287$ kJ/kg, $k = 1.40$)

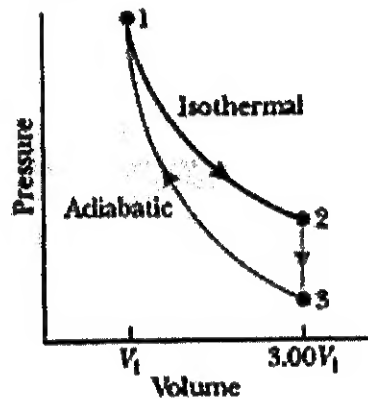


Figure Q1

- Determine the mass of the air. [2 marks]
- Determine P_2 , P_3 and T_3 . [6 marks]
- Calculate the heat transfer per unit mass, q , work output per unit mass, w , and change of internal energy per unit mass, Δu , for each process in kJ/kg. [10 marks]
- Calculate the net heat transfer per unit mass, q_{net} , and net work output per unit mass, w_{net} , for whole process in kJ/kg. [2 marks]
- Is the net work output per unit mass of a closed system during a cycle equal to the net heat input per unit mass? Why? [3 marks]
- Is this a power cycle or a refrigeration cycle? Explain your answer. [2 marks]

Question 2

- Complete the blank cells in the following table of properties of water. In the last column describe the condition of water as compressed liquid, saturated liquid, saturated liquid- vapor mixture, saturated vapor, superheated vapor, or insufficient information; and, if applicable, give the quality.

P , kPa	T , °C	v , m ³ /kg	u , kJ/kg	Quality, x	Condition description
100	70				
200					saturated vapor
20,000	40				
400			3170.5		
	120		2022.6		

[13 marks]

Continued ...

- b) A Carnot heat engine receives heat from a reservoir at 900°C at a rate of 800 kJ/min and rejects the waste heat to the ambient air at 27°C . The entire work output of the heat engine is used to drive a refrigerator that removes heat from the refrigerated space at -5°C and transfers it to the same ambient air at 27°C . Determine
- The maximum rate of heat removal from the refrigerated space and [7 marks]
 - The total rate of heat rejection to the ambient air. [5 marks]

Question 3

A hot-water stream at 80°C enters a mixing chamber with a mass flow rate of 0.5 kg/s where it is mixed with a stream of cold water at 20°C as shown in **Figure Q3**. Assume all the streams are at a pressure of 250 kPa . If it is desired that the mixture leave the chamber at 42°C , determine

- the enthalpy of hot-water stream, [3 marks]
- the enthalpy of cold-water stream, [3 marks]
- the enthalpy of mixture leave the chamber, [3 marks]
- the mass balance equation, [4 marks]
- the energy balance equation, and [8 marks]
- the mass flow rate of the cold-water stream. [4 marks]

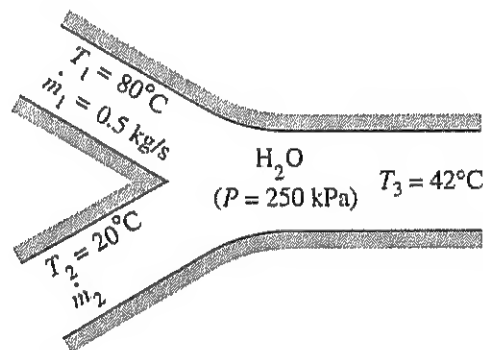


Figure Q3

Continued ...

Question 4

A 0.5-m^3 rigid tank contains refrigerant-134a initially at 200 kPa and 40 percent quality. Heat is transferred now to the refrigerant from a source at 35°C until the pressure rises to 400 kPa. Determine

- a) initial internal energy, [2 marks]
- b) initial entropy, [2 marks]
- c) initial specific volume, [2 marks]
- d) final internal energy, [2 marks]
- e) final entropy, [2 marks]
- f) mass of the refrigerant, [3 marks]
- g) the entropy change of the refrigerant, [3 marks]
- h) the entropy change of the heat source, and [6 marks]
- i) the total entropy change for this process. [3 marks]

Continued ...

Appendix

Uniform State Uniform Flow (Unsteady Flow)

Continuity:

$$(m_2 - m_1) = \sum_i m_i - \sum_e m_e$$

First Law:

$$\begin{aligned} Q_i + W_i + \sum_i m_i \left(h_i + \frac{V_i^2}{2} + gZ_i \right) - Q_e - W_e - \sum_e m_e \left(h_e + \frac{V_e^2}{2} + gZ_e \right) \\ = m_2 \left(u_2 + \frac{V_2^2}{2} + gZ_2 \right) - m_1 \left(u_1 + \frac{V_1^2}{2} + gZ_1 \right) \end{aligned}$$

Second Law:

$$m_2 s_2 - m_1 s_1 = \sum_i m_i s_i - \sum_e m_e s_e + \int_0^t \frac{\dot{Q}_{cv}}{T} dt + S_{2gen}$$

Ideal Gas

Ideal Gas Equations of State

$$\begin{aligned} Pv &= RT \\ dh &= C_p dT \\ du &= C_v dT \end{aligned}$$

Specific Heats and Ideal Gas Constants

$$\begin{aligned} C_p - C_v &= R \\ \frac{C_p}{C_v} &= k \end{aligned}$$

Entropy Relationships

$$\begin{aligned} s_2 - s_1 &= C_v \ln \frac{T_2}{T_1} + R \ln \frac{v_2}{v_1} \quad \text{if constant } C_v \\ &= C_p \ln \frac{T_2}{T_1} - R \ln \frac{P_2}{P_1} \quad \text{if constant } C_p \\ &= s_{T_2}^0 - s_{T_1}^0 - R \ln \frac{P_2}{P_1} \quad \text{otherwise} \end{aligned}$$

For polytropic process

$$PV^n = \text{constant}, \quad TV^{n-1} = \text{constant}, \quad TP^{\left(\frac{n}{n-1}\right)} = \text{constant}$$

$$\begin{aligned} {}_1W_2 &= \frac{P_1 V_1 - P_2 V_2}{n-1} = \frac{mR(T_1 - T_2)}{n-1}, \quad n \neq 1 \\ &= P_1 V_1 \ln \left(\frac{V_2}{V_1} \right), \quad n = 1 \end{aligned}$$

End of Paper.